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Turbulent spots and waves in a model for plane Poiseuille flow

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The structure of a turbulent spot in plane Poiseuille flow is investigated using a model derived from the Navier–Stokes equations through a Galerkin method. The mean profile of the streamwise velocity inside the turbulent spot has the characteristic flat profile of a turbulent Poiseuille flow. The waves developing at the wing tips of the spot have an asymmetric streamwise velocity with respect to the channel centerline, whereas their associated wall-normal velocity component is symmetric. On the outskirts of the spot, a large-scale flow occupying the full gap between the plates is observed. It is characterized by a streamwise inflow toward the spot and a spanwise outflow from the spot. A detailed comparison with the numerical simulations and the experiments in the literature shows that these results are in fair agreement with the main features of the transitional plane Poiseuille flow.

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I. INTRODUCTION

Plane Poiseuille flow (pPf), shear flow between two fixed parallel plates driven by a pressure gradient, experiences a transition to turbulence marked by the nucleation and growth of *turbulent spots*, i.e., patches of turbulent flow scattered amidst laminar flow, when the Reynolds number R (based on the half-channel height h and the centerline velocity U_{cl}) exceeds a certain threshold R_g [e.g., $R_g \sim 1000$ for Carlson *et al.* (Ref. 1) and $R_g \sim 1100$ for Alavyoon *et al.* (Ref. 2)].

This kind of transition is not restricted to the pPf case but also occurs in other shear flows such as plane Couette flow³ and boundary layer flows.⁴ Despite a large body of numerical^{5–7} and laboratory^{1,3} experiments, many questions regarding such transition remain unanswered, such as the mechanisms involved in the growth of turbulent spots^{8,9} and in the self-sustainment of the turbulent state.¹⁰

In their experimental investigation of the transitional plane Poiseuille flow, Carlson *et al.*¹ noted that the spot has a central turbulent area, in front of which there is a disturbed but not turbulent region, whereas at the wing tips of the spot there are oblique waves.

The origin of these waves was investigated by Li and Widnall,¹¹ who modeled the spot by a moving patch of Reynolds stress. Their numerical simulations have shown that spatially damped oblique waves, resembling those observed at the front of a turbulent spot, are generated. Furthermore, the nature of these waves and their effect on the dynamics of the spot were studied numerically by Henningson *et al.*¹² Due to the presence of the spot, the mean spanwise profile is inflectional, and oblique waves may grow and then break down into turbulence. However, the linear growth rate of these waves calculated by Henningson¹³ is too small compared to the observed one. Therefore, he suggested that the waves attain their large growth rate by some additional mechanisms. Moreover, the characteristics of these waves,

measured by Henningson and Alfredsson¹⁴ using hot film anemometry, have been found to be in fair agreement with the theoretical Tollmien–Schlichting waves, i.e., the least stable mode of the Orr–Sommerfeld equation, as conjectured by Carlson *et al.*¹

The role of these waves on the spreading of the spot was addressed by Alavyoon *et al.*² By pointing out the absence of waves at the wing tips of the spots in the boundary layer flow, they concluded that the waves would be of no importance for the spreading itself, if the same spreading mechanism is at work in both plane Poiseuille and boundary layer flows.

Therefore, despite a large body of experiments and well described results, the dynamics of turbulent spots in pPf remains poorly understood and many questions, such as the nature of the observed waves and the mechanisms of maintenance of the turbulence, remain open.

An attempt to tackle such questions in the case of plane Couette flow led us to derive a model in terms of three partial differential equations.¹⁵ Such a model has brought some elements of understanding to these problems, such as the nature of the flow on the outskirts of a turbulent spot¹⁶ and the spreading mechanism.¹⁷

This paper is devoted to the derivation and the study of such a model for the plane Poiseuille flow.

The outline of the paper is as follows. The model is first derived in Sec. II. Then some numerical results on the dynamics of turbulent spots are presented in Sec. III. The flow outside and inside the turbulent domain is analyzed, and waves at the wing tips are observed. The main results of this paper are assessed in Sec. IV.

II. THE MODEL

The Navier–Stokes equation and continuity condition for an incompressible flow read

$$\partial_t \mathbf{v} + \mathbf{v} \cdot \nabla \mathbf{v} = -\nabla p + \nu \nabla^2 \mathbf{v}, \quad (1)$$

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$$\nabla \cdot \mathbf{v} = 0, \quad (2)$$

with $\mathbf{v} \equiv (u, v, w)$, where u is the streamwise (x), v is the wall-normal (y), and w is the spanwise (z) velocity component, p is the pressure, and ν is the kinematic viscosity. ∇^2 denotes the three-dimensional Laplacian. In the following, we use dimensionless quantities. Lengths are scaled with the half-channel height h , and the centerline velocity U_{cl} stands for the velocity scale. Hence the velocity profile of the base flow, generated by a constant pressure gradient, is $U(y) = U_b(1 - y^2)$ for $y \in [-1, 1]$, where $U_b = 1$ and the Reynolds number is $R = U_{cl}h/\nu$. Equations (1) and (2) are further developed for the perturbations (u', v', w', p') to the laminar flow and read

$$\begin{aligned} & \partial_t u' + (\partial_y u' - \partial_x v')v' + (\partial_z u' - \partial_x w')w' \\ &= -\partial_x p' - U\partial_x u' - v'\frac{d}{dy}U + R^{-1}\nabla^2 u', \end{aligned} \quad (3)$$

$$\begin{aligned} & \partial_t v' + (\partial_x v' - \partial_y u')u' + (\partial_z v' - \partial_y w')w' \\ &= -\partial_y p' - U\partial_x v' + R^{-1}\nabla^2 v', \end{aligned} \quad (4)$$

$$\begin{aligned} & \partial_t w' + (\partial_x w' - \partial_z u')u' + (\partial_y w' - \partial_z v')v' \\ &= -\partial_z p' - U\partial_x w' + R^{-1}\nabla^2 w', \end{aligned} \quad (5)$$

$$0 = \partial_x u' + \partial_y v' + \partial_z w', \quad (6)$$

where the nonlinear terms have been rewritten using the rotational form (see Appendix A).

A. Expansions of the velocity components

The Galerkin method is a special case of a weighted residual method. It consists of the separation of the in-plane (x, z) and the wall-normal (y) coordinates by expanding the perturbations (u', v', w', p') onto a complete orthogonal basis of y -dependent functions satisfying the boundary conditions with amplitudes dependent on (x, z, t) . The equations of motion are then projected onto the same functional basis. The main modeling step is then performed when truncating these expansions at a low order to get a consistent and closed system governing the retained amplitudes. The projections are performed by taking the canonical scalar product $\langle \cdot, \cdot \rangle$ defined by

$$\langle f, g \rangle = \int_{-1}^{+1} f(y)g(y)dy. \quad (7)$$

The no-slip boundary conditions for the wall-normal velocity component are

$$v'|_{y=\pm 1} = \partial_y v'|_{y=\pm 1} = 0 \quad (8)$$

obtained by combining the continuity equation (6) to the conditions

$$u'|_{y=\pm 1} = w'|_{y=\pm 1} = 0. \quad (9)$$

Hence, the wall-normal velocity is expanded as

$$v'(x, z, y, t) = \sum_{n \geq 1} V_n(x, z, t) S_n(y)$$

where for an integer n , $S_n(y) = (1 - y^2)^2 Q_n(y)$, where Q_n is an arbitrary polynomial. The first two polynomials are

$$S_1(y) = Cy(1 - y^2)^2 \quad \text{and} \quad S_2(y) = B(1 - y^2)^2,$$

where B and C are constants. In the same way, the polynomials R_n for the in-plane velocity (u', w') have the form $R_n(y) = (1 - y^2)T_n(y)$, where T_n is an arbitrary polynomial. The first polynomial is $R_0(y) = (1 - y^2)$. The second and the third ones are determined by the continuity equation through $R_n(y) \propto d/dy S_n(y)$, $n = 1, 2$, and read

$$R_1(y) = B(1 - y^2)(5y^2 - 1) \quad \text{and} \quad R_2(y) = Fy(1 - y^2).$$

The next step in the modeling is to introduce the truncated expansions into Eqs. (3)–(6) and to project over each polynomial using the scalar product (7).

The parity properties of the polynomials guarantee the orthogonality of the different contributions. Hence, the contribution of R_2 is separated from the contributions of R_0 and R_1 . However, R_0 and R_1 have the same parity and hence their contributions are not separated. To remedy to this, we use the Gram–Schmidt orthogonalization method to construct R_0 , which is orthogonal to R_1 . With the polynomial $R_0(y) = A(1 - y^2)(1 + 3y^2)$, we have $\langle R_0, R_1 \rangle = 0$ and the considered expansions read

$$u' = U_0(x, z, t)R_0(y) + U_1(x, z, t)R_1(y) + U_2(x, z, t)R_2(y), \quad (10)$$

$$v' = V_1(x, z, t)S_1(y) + V_2(x, z, t)S_2(y), \quad (11)$$

$$w' = W_0(x, z, t)R_0(y) + W_1(x, z, t)R_1(y) + W_2(x, z, t)R_2(y), \quad (12)$$

$$p' = P_0(x, z, t)R_0(y) + P_1(x, z, t)R_1(y) + P_2(x, z, t)R_2(y), \quad (13)$$

with the polynomials

$$R_0(y) = A(1 - y^2)(1 + 3y^2),$$

$$R_1(y) = B(1 - y^2)(5y^2 - 1),$$

$$R_2(y) = Fy(1 - y^2),$$

$$S_1(y) = Cy(1 - y^2)^2,$$

$$S_2(y) = B(1 - y^2)^2,$$

plotted in Fig. 1. The normalization constants are $A^2 = 105/256$, $B^2 = 315/256$, $C^2 = 3465/256$, and $F^2 = 105/16$. The pressure components P_0 , P_1 , and P_2 introduce themselves as the Galerkin projection of the pressure p' on R_0 , R_1 , and R_2 , respectively.

Next, by inserting the expansions (10)–(12) in the continuity equation (6) and projecting, respectively, on the polynomials R_0 , R_1 , and R_2 , we get the three equations

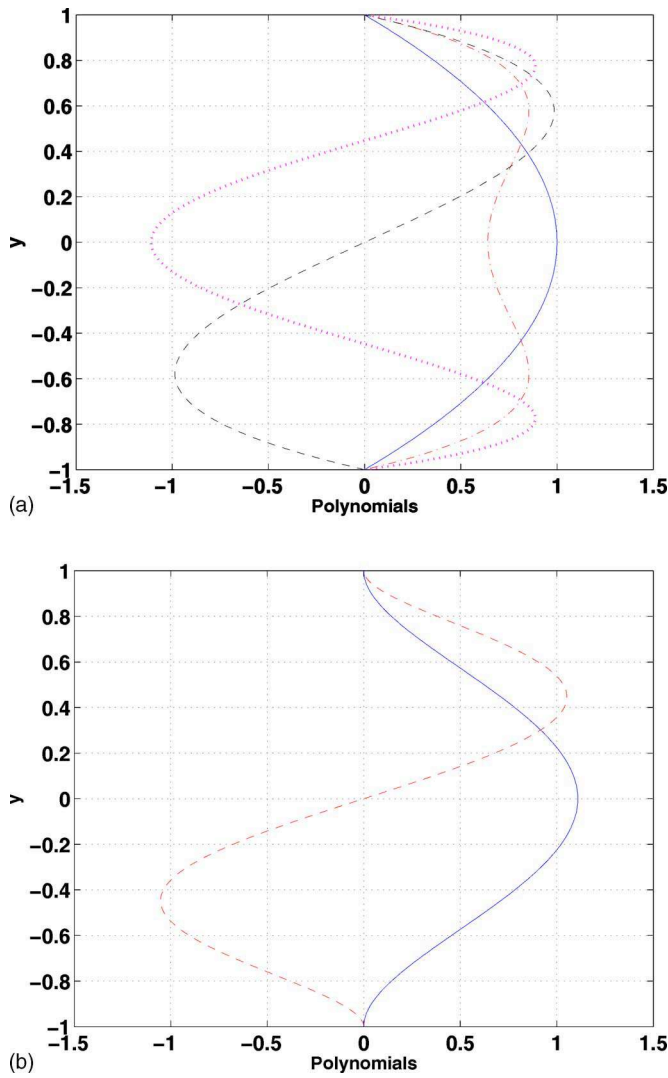


FIG. 1. (Color online) Profiles of the basis functions used in the derivation of the model. (a) Polynomials for the in-plane velocity. R_0 with dash-dotted (red) line, R_1 in dotted (magenta), R_2 in dashed (black), and the base flow $U(y) = U_b(1-y^2)$ in solid (blue). (b) Polynomials for the wall-normal velocity. S_1 in dashed (red) and S_2 in solid (blue).

$$\partial_x U_0 + \partial_z W_0 = 0, \quad \partial_x U_1 + \partial_z W_1 = \beta_1 V_1, \quad (14)$$

$$\partial_x U_2 + \partial_z W_2 = \beta_2 V_2,$$

where $\beta_1 = \sqrt{11}$ and $\beta_2 = \sqrt{3}$. Then from the projection of the Navier–Stokes equations (3)–(5) on the expansions (10)–(13), we get the equations of the amplitudes.

For U_0 and W_0 , we have

$$\begin{aligned} \partial_t U_0 + N_{U_0} = & -\partial_x P_0 - b_1 U_b \partial_x U_0 + b_2 U_b \partial_x U_1 + \beta_1 b_3 U_b V_1 \\ & - b_4 U_0 / R + R^{-1}(\Delta_2 - \gamma_0) U_0, \end{aligned} \quad (15)$$

$$\begin{aligned} \partial_t W_0 + N_{W_0} = & -\partial_z P_0 - b_1 U_b \partial_x W_0 + b_2 U_b \partial_x W_1 - b_4 W_0 / R \\ & + R^{-1}(\Delta_2 - \gamma_0) W_0, \end{aligned} \quad (16)$$

with $\gamma_0 = \frac{11}{2}$, $\Delta_2 = \partial_{xx} + \partial_{zz}$, and the nonlinear terms, noted N_{U_0} and N_{W_0} , are given in Appendix B. In the same way, we obtain the equations of U_1 , W_1 , and V_1 :

$$\begin{aligned} \partial_t U_1 + N_{U_1} = & -\partial_x P_1 + b_2 U_b \partial_x U_0 - b_1 U_b \partial_x U_1 + U_{b1} V_1 / \beta_1 \\ & - b_4 U_0 / R + R^{-1}(\Delta_2 - \gamma_1) U_1, \end{aligned} \quad (17)$$

$$\begin{aligned} \partial_t W_1 + N_{W_1} = & -\partial_z P_1 + b_2 U_b \partial_x W_0 - b_1 U_b \partial_x W_1 - b_4 W_0 / R \\ & + R^{-1}(\Delta_2 - \gamma_1) W_1, \end{aligned} \quad (18)$$

$$\partial_t V_1 + N_{V_1} = -\beta_1 P_1 - b_5 U_b \partial_x V_1 + R^{-1}(\Delta_2 - \beta_1^2) V_1, \quad (19)$$

with $\gamma_1 = \frac{45}{2}$, and the equations of U_2 , W_2 , and V_2 :

$$\begin{aligned} \partial_t U_2 + N_{U_2} = & -\partial_x P_2 + \frac{U_b}{\beta_2} V_2 - \frac{2}{3} U_b \partial_x U_2 \\ & + R^{-1}(\Delta_2 - \gamma_2) U_2, \end{aligned} \quad (20)$$

$$\partial_t W_2 + N_{W_2} = -\partial_z P_2 - \frac{2}{3} U_b \partial_x W_2 + R^{-1}(\Delta_2 - \gamma_2) W_2, \quad (21)$$

$$\partial_t V_2 + N_{V_2} = -\beta_2 P_2 - \frac{10}{11} U_b \partial_x V_2 + R^{-1}(\Delta_2 - \beta_2^2) V_2, \quad (22)$$

with $\gamma_2 = \frac{21}{2}$. The nonlinear terms N_{U_1} , N_{W_1} , N_{V_1} , N_{U_2} , N_{W_2} , and N_{V_2} are given in Appendix B. Then, to eliminate the pressures P_0 , P_1 , and P_2 in Eqs. (15) and (16), Eqs. (17)–(19) and Eqs. (20)–(22), respectively, we introduce stream functions ($\Psi_0(x, z, t)$, $\Psi_1(x, z, t)$, $\Psi_2(x, z, t)$) and velocity potentials ($\Phi_1(x, z, t)$, $\Phi_2(x, z, t)$) that satisfy the continuity equations,

$$U_0 = -\partial_z \Psi_0, \quad W_0 = \partial_x \Psi_0, \quad (23)$$

$$U_1 = \partial_x \Phi_1 - \partial_z \Psi_1, \quad W_1 = \partial_z \Phi_1 + \partial_x \Psi_1, \quad \beta_1 V_1 = \Delta_2 \Phi_1, \quad (24)$$

$$U_2 = \partial_x \Phi_2 - \partial_z \Psi_2, \quad W_2 = \partial_z \Phi_2 + \partial_x \Psi_2, \quad \beta_2 V_2 = \Delta_2 \Phi_2. \quad (25)$$

The equation governing Ψ_0 (Ψ_1 and Ψ_2) is obtained by cross-differentiating and subtracting the equations for the velocity components (15) and (16) [(17) and (18) and (20) and (21)] (the rotational part). Then, to derive the equation for the velocity potential Φ_1 (Φ_2), we take the divergence of Eqs. (17) and (18) [(20) and (21)], which yields an equation for the pressure P_1 (P_2), which is next used with Eq. (19) [Eq. (22)] to determine the potential part of the velocity field accounted for by the field Φ_1 (Φ_2). Finally, the five equations of the model are

$$\begin{aligned} [\partial_t - R^{-1}(\Delta_2 - \gamma_0)] \Delta_2 \Psi_0 = & -b_1 \partial_x \Delta_2 \Psi_0 + b_2 \partial_x \Delta_2 \Psi_1 \\ & - b_3 \partial_z \Delta_2 \Phi_1 - b_4 \Delta_2 \Psi_1 / R \\ & + (\partial_z N_{U_0} - \partial_x N_{W_0}), \end{aligned} \quad (26)$$

$$\begin{aligned} [\partial_t - R^{-1}(\Delta_2 - \gamma_1)] \Delta_2 \Psi_1 = & -b_1 \partial_x \Delta_2 \Psi_1 + b_2 \partial_x \Delta_2 \Psi_0 \\ & - b_4 \Delta_2 \Psi_0 / R - \partial_z \Delta_2 \Phi_1 / \beta_1^2 \\ & + (\partial_z N_{U_1} - \partial_x N_{W_1}), \end{aligned} \quad (27)$$

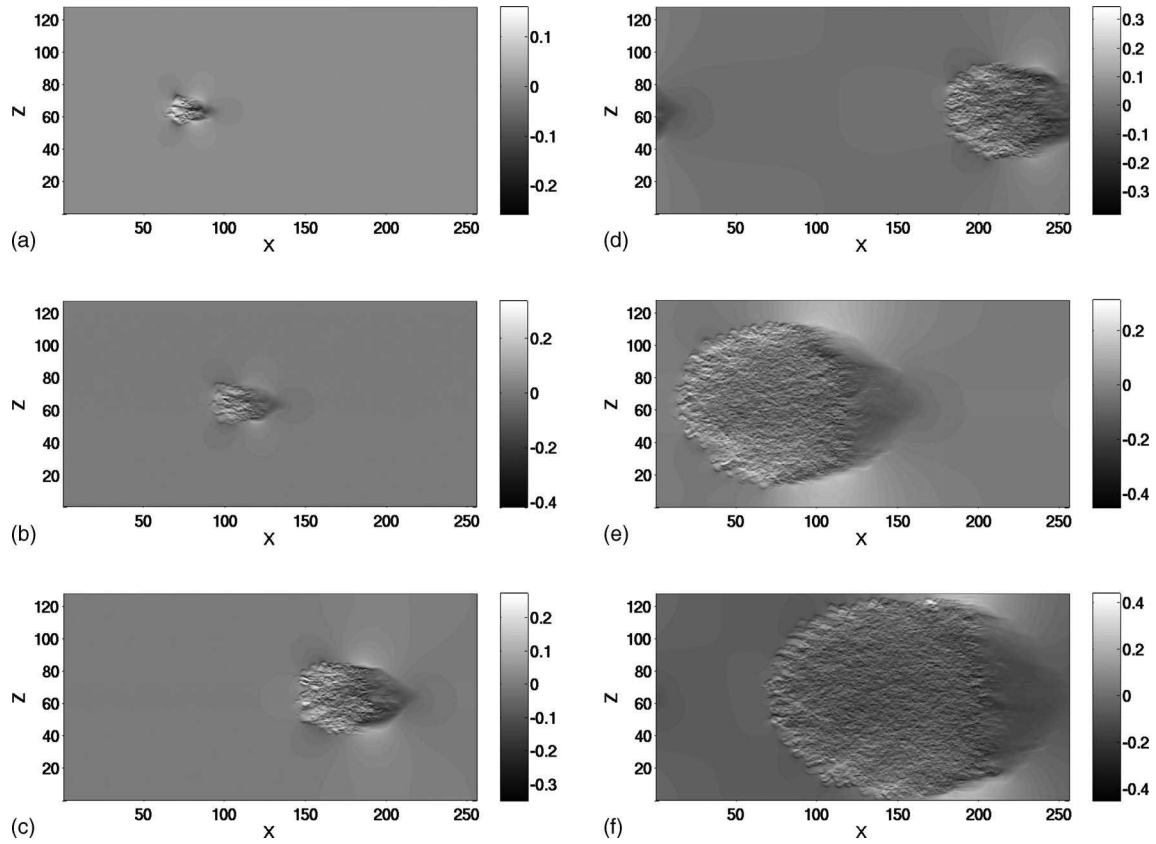


FIG. 2. Evolution of a turbulent spot depicted by the streamwise velocity U_0 . $R=900$. From top to bottom and left to right: $t=64, 112, 208, 272, 448$, and 560 (enhanced online).

$$\begin{aligned}
 & [\partial_t - R^{-1}(\Delta_2 - \beta_1^2)](\Delta_2 - \beta_1^2)\Delta_2\Phi_1 \\
 &= \beta_1^2(\partial_x N_{U_1} + \partial_z N_{W_1}) - \beta_1\Delta_2 N_{V_1} + \frac{23\beta_1^2}{2R}\Delta_2\Phi_1 \\
 &+ (7 - b_5\Delta_2)\partial_x\Delta_2\Phi_1, \quad (28)
 \end{aligned}$$

$$\begin{aligned}
 & [\partial_t - R^{-1}(\Delta_2 - \gamma_2)]\Delta_2\Psi_2 = -\frac{2}{3}\partial_x\Delta_2\Psi_2 - \partial_z\Delta_2\Psi_2/\beta_2^2 \\
 &+ (\partial_z N_{U_2} - \partial_x N_{W_2}), \quad (29)
 \end{aligned}$$

$$\begin{aligned}
 & [\partial_t - R^{-1}(\Delta_2 - \beta_2^2)](\Delta_2 - \beta_2^2)\Delta_2\Phi_2 \\
 &= \beta_2^2(\partial_x N_{U_2} + \partial_z N_{W_2}) - \beta_2\Delta_2 N_{V_2} + \frac{15\beta_2^2}{2R}\Delta_2\Phi_2 \\
 &+ \left(1 - \frac{10}{11}\Delta_2\right)\partial_x\Delta_2\Phi_2. \quad (30)
 \end{aligned}$$

III. NUMERICAL SIMULATIONS

A standard Fourier pseudospectral method with periodic boundary conditions in the streamwise (x) and spanwise (z) directions has been used for the integration of the equations of the model (26)–(30). A second-order Adams–Bashforth scheme is used for the advancement in time. Throughout the paper, numerical simulations are performed in a computational box with streamwise and spanwise lengths $L_x \times L_z = 256 \times 128$. The spatial resolution is $dx=dz=0.125$ and the

time step is $dt=0.01$. For the considered value of the Reynolds number ($R=900$), the obtained results are quite the same for higher resolutions ($dx=dz=0.1, dx=dz=0.05$) and smaller time step ($dt=0.001$).

For the initial condition, we take the localized functions,

$$\Psi_n(x, z, t=0) = A \exp^{-(x^2+z^2)/\sigma}, \quad n=0, 1, 2,$$

$$\Phi_n(x, z, t=0) = 0, \quad n=1, 2,$$

where $A=1$ is an amplitude and $\sigma=2$ is related to the size of the initial turbulent domain.

For $R \leq 850$, all the initial turbulent states decayed, whereas for $R \geq 900$, growing spots have been obtained. Hence, the stability threshold R_g for this model is between 850 and 900. We did not determine the value of R_g with more precision, since the aim of this paper is to study the features of the turbulent spot and to focus on the mechanisms at work.

The spatiotemporal evolution of a turbulent spot can be illustrated by one of its velocity components.¹⁹ From a qualitative point of view, the evolution depicted by the total streamwise velocity (base flow + perturbations) at the channel centerline is similar to that depicted by the component U_0 , plotted at different times in Fig. 2. The interior of the spots shows x -elongated patches of U_0 with z -alternating sign. The physical interpretation of these patches is postponed for future work. One can already note that a positive

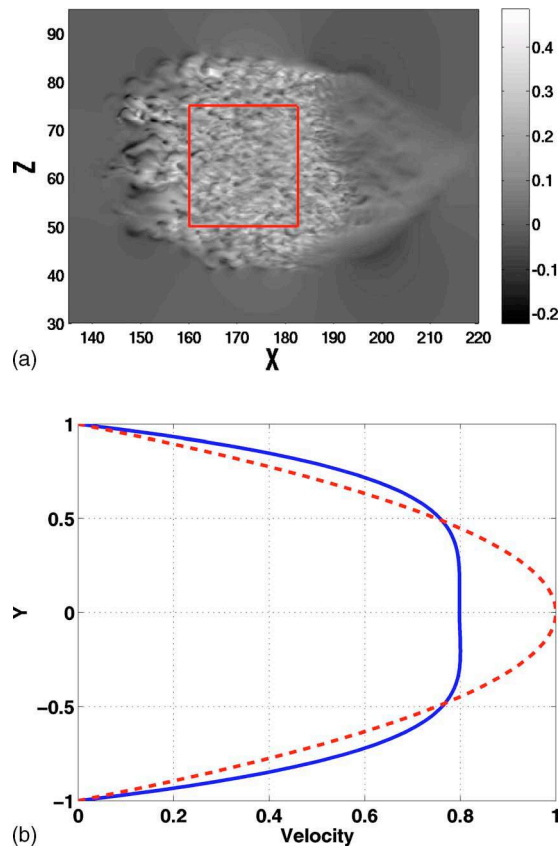


FIG. 3. (Color online) (a) Spatial distribution of U_1 . In the turbulent region limited by the (red) box, U_1 is mainly positive. (b) The mean profile of the total streamwise velocity (in solid blue). The laminar parabolic profile is shown with the dashed (red) line.

U_0 represents a high-speed streak since $U(y) + U_0 R_0(y) \geq U(y)$ and a negative U_0 represents a low-speed streak since $U(y) + U_0 R_0(y) \leq U(y)$.

The general shape and the evolution of the spot compare well with the visualizations of both Carlson *et al.*¹ and Alavyoon *et al.*² At an early stage, the spot has an arrowhead front part ($t=112$). Then, as time proceeds, a disturbed laminar-like region develops at its leading edge (e.g., $x \geq 200$ at $t=560$) and the spot gets a turbulent crescent shape.

In the following, some characteristics of the turbulent spot are presented, showing a qualitative consistency between previous experimental results and our own numerical simulations of the model.

A. Mean profiles

Figure 3 shows the spot at $t=208$ depicted by the velocity component U_1 . The turbulent region is clearly separated from the disturbed laminar-like region in front of the spot (for $x \geq 190$). This turbulent region is limited by the red-box and we define the mean value of any velocity component Z over the surface S of this box by

$$\bar{Z} = \int_S Z dx dz / S.$$

The mean profile of the total streamwise velocity $U(y) + \bar{u}' = U(y) + \bar{U}_0 R_0 + \bar{U}_1 R_1 + \bar{U}_2 R_2$ is given in Fig. 3(b). It

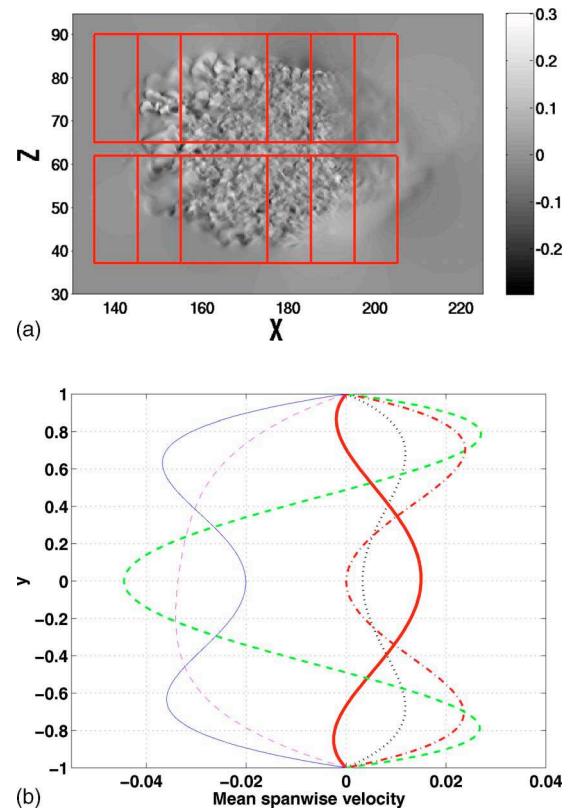


FIG. 4. (Color online) (a) The distribution of the spanwise velocity W_1 . The spot is split into small (arbitrary) regions limited by (red) boxes. There are six boxes for $z \geq 64$ and six others for $z \leq 64$. (b) The mean value of the total spanwise velocity in each bottom box ($z \leq 64$): From left to right: box number (1) with a dotted (black) line, (2) with a thick solid (red) line, (3) with a thin solid (blue) line, (4) with a thin dashed (magenta) line, (5) with a thick dashed (green) line, and (6) with a thin dash-dotted (red) line.

is similar to the turbulent profile of pPf and this is mainly due to the positiveness of $\bar{U}_1$ within the turbulent region, $\bar{U}_1 = 0.1650$, whereas the mean values of the other velocity components are one order smaller, $\bar{U}_0 = -0.029$ and $\bar{U}_2 = -0.0021$. Hence, the spot behaves as a localized region where the laminar flow is decelerated near the center [$U(y) + \bar{U}_1 R_1(y) \leq U(y)$] and accelerated near the walls [$U(y) + \bar{U}_1 R_1(y) \geq U(y)$].

Furthermore, Fig. 4 shows the same spot depicted by another velocity component, W_1 . The mean values of the total spanwise velocity [Eq. (12)] over the boxes shown in Fig. 4(a) are given in Figs. 4(b) and 5(a). At different locations, the spanwise velocity has inflectional profiles similar to those used in Ref. 13 and shown, through a linear stability calculation, to be potentially unstable to oblique waves. Such waves are now studied.

B. The waves at the wing tips of the spot

One of the most interesting features of the spots in pPf is the presence of oblique waves at the wing tips of the spots. As shown in Fig. 5(b), such waves are easily observed by reporting the wall-normal velocity on the channel centerline. At this y location, the wall-normal velocity given by Eq. (11)

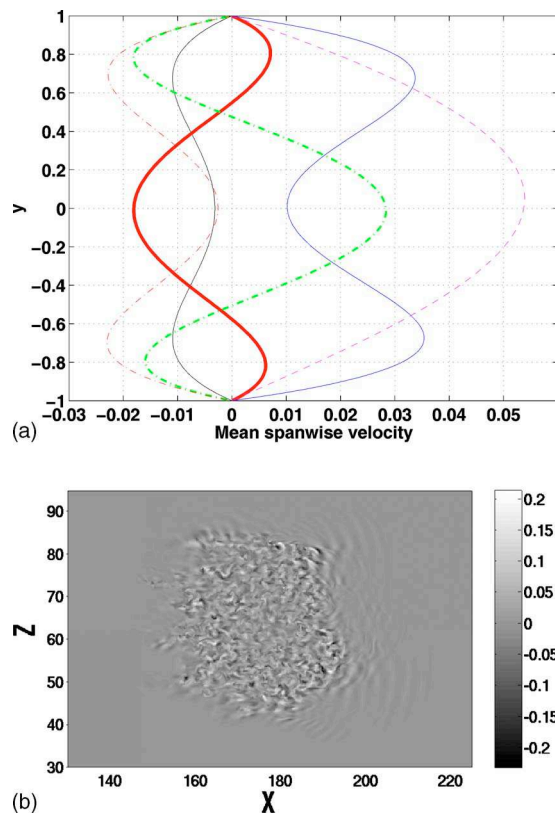


FIG. 5. (Color online) (a) The mean value of the total spanwise velocity in each upper box ($z \geq 64$), shown in Fig. 4. Same color convention as in Fig. 4(b). (b) The spatial distribution of V_2 . At the channel centerline, the total wall-normal velocity is equal to BV_2 . Oblique waves are present at the wing tips of the spot (enhanced online).

reduces to BV_2 . These waves can also be represented by the in-plane velocity component associated with V_2 , i.e., W_2 and U_2 , and are displayed in Fig. 6.

The profiles of U_2 , W_2 , and V_2 as a function of the streamwise coordinate (x) for $z=44$ and $z=86$ are given in Fig. 7. These plots clearly point out the wavy character of the velocity components U_2 , W_2 , and V_2 at the wing tip regions. On the one hand, the wavelength of these waves is about $5h$ and is of the same order as the one measured by Alavyoon *et al.*,² where it is about two channel heights (i.e., $4h$). On the other hand, regarding the magnitude of these waves, Henningson¹³ found an amplitude of 15% of U_{cl} for the wall-normal velocity component at the wing tips of the spot. In the present results, the amplitude is about 5% of U_{cl} ($B \sim 1.1$).

Furthermore, Henningson and Alfredsson¹⁴ have investigated these wave packets by hot-film anemometry and have found that the streamwise velocity disturbance associated with these waves is antisymmetric with respect to the channel centerline (Fig. 7 in Ref. 14). This noteworthy point is in accordance with our results, since the y dependence of the streamwise velocity U_2 , associated with these waves, is odd in y and is given by the polynomial $R_2(y) = Fy(1 - y^2)$.

As a conclusion, the waves packets located at the wing tips of the turbulent spot have a wall-normal velocity with a symmetric y distribution and streamwise and spanwise velocities with an antisymmetric y distribution. Thus, they have

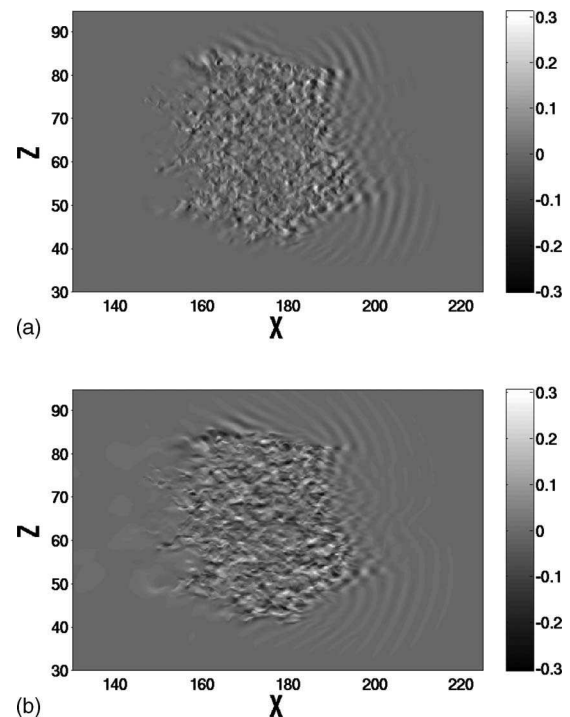


FIG. 6. The spatial distributions of the spanwise velocity W_2 (a) and of the streamwise velocity U_2 (b) associated with V_2 show waves at the wing tips of the spot. Not all the points are represented (enhanced online).

the same properties as the waves investigated by Henningson and Alfredsson¹⁴ and Henningson and Kim.¹⁸

C. Large-scale flow around the spot

Studying the spanwise velocity at the channel centerline, Henningson and Kim¹⁸ observed that there is a mean motion out from the spot toward the wing tips and that this outward motion continues in the laminar flow outside the wing tips, whereas there is a motion toward the spot downstream of the turbulent area. Near the walls, they also observed that the spanwise velocity exhibits the same feature as it does at the midplane. This particular distribution indicates that the flow around the spot is characterized by an outflow from the spot in the spanwise direction extending all over the gap and by a streamwise inflow toward the spot.

Within our modeling approach, this inflow is represented by the streamwise velocity component $U_0 R_0(y)$ and this outflow by the spanwise component $W_0 R_0(y)$. In fact, as shown in Fig. 8, on the leading edge of the spot ($x \geq 190$), U_0 is mainly negative, whereas on the trailing edge ($x \leq 150$) it is mainly positive. Hence, this distribution points out the inflow character of the streamwise component U_0 . To this inflow motion toward the spot corresponds an outflow motion represented by two regions where W_0 is mainly positive (for $z \geq 70$) and negative (for $z \leq 50$), as shown in Fig. 8.

By combining these features of the spanwise outflow and the streamwise inflow, we obtain a quadrupolar flow, shown in Fig. 9(a), similar to that observed around a turbulent spot in the plane Couette flow.⁷ The motion toward the spot downstream of the turbulent area [i.e., $x \geq 190$, in Fig. 9(a)] is more important than the motion toward the spot upstream.

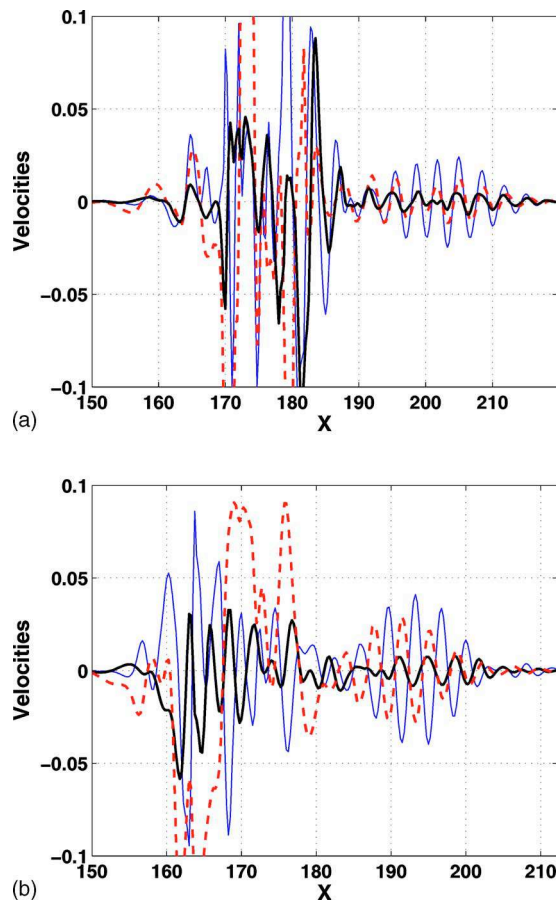


FIG. 7. (Color online) The profiles of U_2 (in dashed red), W_2 (in thin blue), V_2 (in thick black) along a streamwise line for $z=44$ (a) and $z=86$ (b). The wave packets are clearly observed for $x \in [190, 215]$ (a) and $x \in [180, 210]$ (b).

This quadrupolar structure can be already seen from the distribution of the stream function Ψ_0 associated with the flow (U_0, W_0) . This distribution, displayed in Fig. 9(b), is characterized by four lobes with alternating sign. Each lobe corresponds to a region of recirculation extending outside the spot.

The origin of this quadrupolar large-scale flow in the case of pCf was traced back to the positiveness of the Reynolds stress within the turbulent region.¹⁶ This may provide the starting point for a similar investigation in the present case.

D. Coherent structures

The interior structure of the turbulent domain is now investigated. A particular interest is given to the streamwise vortices, which play an important role in the energy production and hence in the sustenance of the turbulent state.

First, the Reynolds–Orr equation governs the time evolution of the perturbations energy $E(t) = \frac{1}{2} \int_{\mathcal{V}} (u'^2 + v'^2 + w'^2) d\mathcal{V}$, where $\mathcal{V}$ is the volume of the domain. If P denotes the energy production issued from the interaction of the perturbation with the base flow $U(y)$, $P = \int_{\mathcal{V}} -u'v' \frac{dU}{dy} d\mathcal{V}$, and if D is the dissipation due to viscous effects, then this equation is written as $\frac{d}{dt}E = P - D$. The product $-u'v'$ is the Reynolds

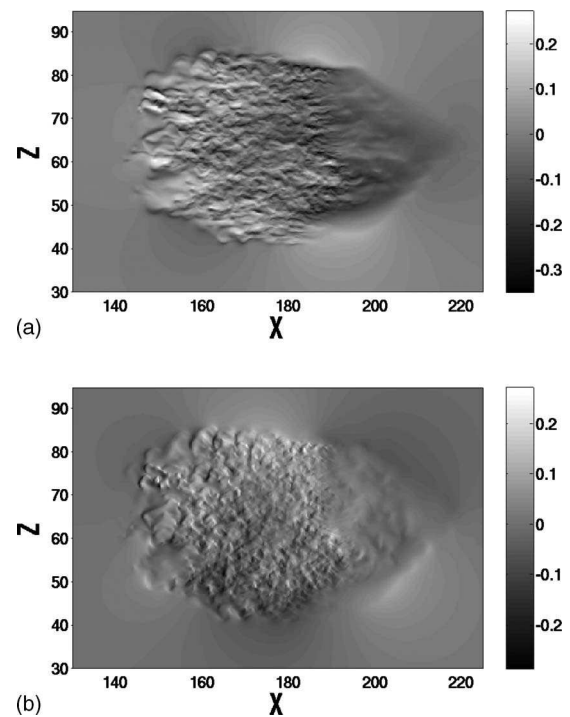


FIG. 8. The spatial distribution of U_0 (a) represents an inflow toward the spot, whereas W_0 (b) represents an outflow from the spot. Outside the spot, the in-plane flow (U_0, W_0) is quadrupolar, as shown in Fig. 9.

stress component associated with the energy production. Within our modeling approach, by integrating over the gap, the production reads

$$P = \int_S \frac{5}{\sqrt{33}} U_0 V_1 + \frac{1}{\sqrt{11}} U_1 V_1 + \frac{1}{\sqrt{3}} U_2 V_2 dS, \quad (31)$$

where S is the surface of the domain.

Second, the redistribution of the base flow by the wall-normal velocity associated with the streamwise vortices generates the streaks. This mechanism, called the lift-up effect, is represented by the linear term $-v' \frac{dU}{dy}$ in Eq. (3). The linear term $\beta_1 b_3 U_b V_1$ in the U_0 equation comes from the Galerkin projection over R_0 of the lift-up term and accounts for the generation of the streak U_0 by the wall-normal velocity V_1 . Regions where the lift-up effect occurs are hence characterized by positive Reynolds stress $U_0 V_1$.

The inspection of the full turbulent domain, localized or filling the whole computational box, shows that the streamwise vortices are numerous. They are easily observed by monitoring the wall-normal velocity V_1 , as shown in Fig. 10. The negative crescent contour of V_1 represents a flow going from the bottom wall ($y=-1$) to the channel centerline ($y=0$) since $V_1 S_1(y) \geq 0$. Then, it goes back toward the bottom wall (where $V_1 \geq 0$) through the in-plane motion represented by the flow field $(U_1, W_1) R_1(y)$ [Fig. 10(b)]. This particular motion represents a crescent vortex in the half-space $y \in [-1, 0]$, as shown in Fig. 11. Due to the asymmetry of the polynomial S_1 and to the symmetry of the polynomial R_1 , a similar crescent vortex exists in the other half-space $y \in [0, 1]$, but rotating in the opposite sense. The two counter-rotating streamwise vortices forming the legs of the crescent

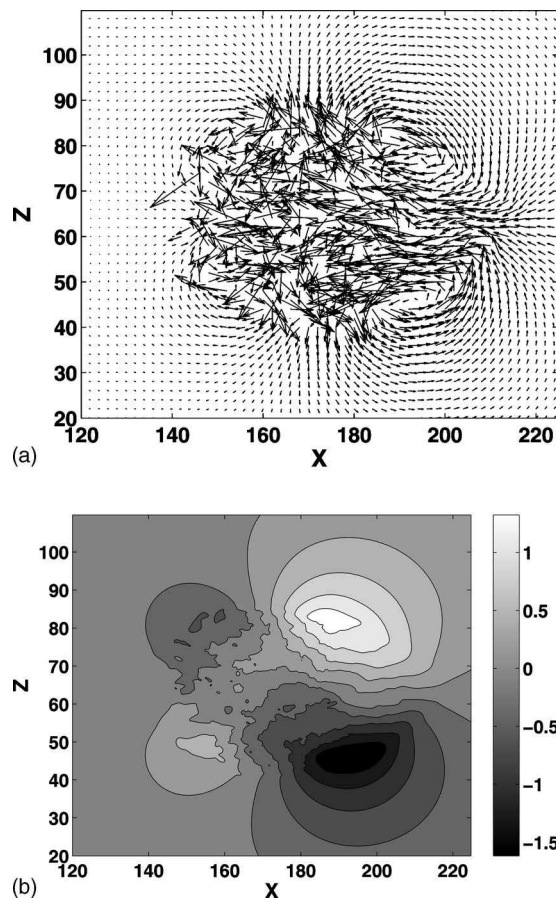


FIG. 9. (a) The flow field (U_0, W_0) exhibits a quadrupolar structure (not all the points are represented). (b) Spatial distribution of the stream function Ψ_0 . Outside the spot, it has four lobes with alternating sign. Each lobe corresponds to a region of large recirculation.

vortex regenerate the streaks by the lift-up effect: regions with positive (negative) V_1 in Fig. 10 correspond to regions of positive (negative) U_0 in Fig. 12. Thus, on each side of the legs, we have positive Reynolds stress $U_0 V_1$ as shown in Fig. 12(b).

Note that the flow field (U_0, W_0) , shown in Fig. 12, has some dipoles, which have been shown in Ref. 17 to be at the origin of the spreading of the turbulent domain through their self-advection.

Following the study of Papavassiliou and Hanratty,²⁰ a comparison between the geometries of the streamwise vortices in the plane Couette and Poiseuille flows for low-Reynolds numbers as well as their generation process will be presented in a forthcoming paper.

IV. CONCLUSION

In this paper, we have derived a model in terms of five partial differential equations for pPf. The numerical simulation of the turbulent spot shows many features in close agreement with the results in the literature, supporting hence the modeling approach.

The spot consists of three distinct regions: a turbulent area, a disturbed area, and a wave area. Inside the turbulent area, the laminar flow is accelerated near the walls, whereas it is decelerated near the center of the channel, getting a flat

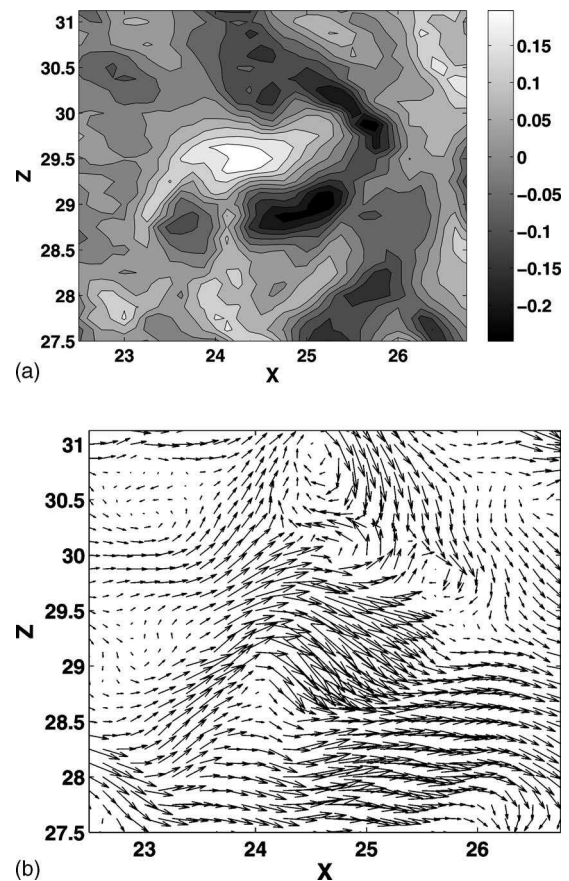


FIG. 10. Spatial distribution of V_1 (a) and the flow field (U_1, W_1) (b). This spatial distribution represents a crescent vortex in each half-space $y \in [-1, 0]$ and $y \in [0, 1]$. A 3D representation is given in Fig. 11. The two legs of the crescent vortex are elongated in the streamwise direction and form the streamwise vortices.

shape typical of the turbulent plane Poiseuille flow. The oblique waves, observed at the wing tip regions, share several features with those studied in the literature, especially their velocity structure. They have a symmetric wall-normal velocity ($V_2 S_2$) and an antisymmetric streamwise velocity ($U_2 R_2$) with respect to the channel centerline. The origin of

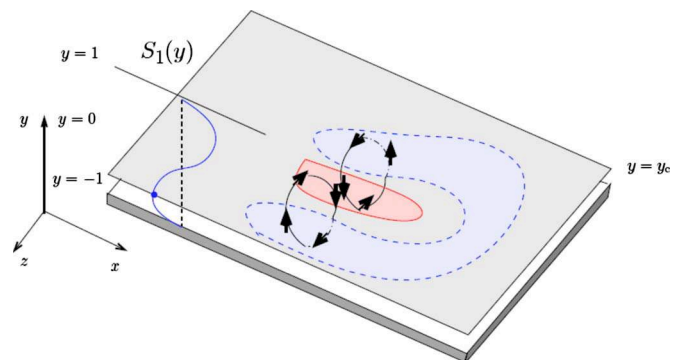


FIG. 11. (Color online) 3D reconstruction of the crescent vortex, depicted by V_1 and (U_1, W_1) in Fig. 10. The distribution of V_1 has a crescent shape in the shaded plane for some $y = y_c \in [0, -1]$. The two circles represent the cross section (z, y) of the streamwise vortices. The polynomial S_1 is plotted by a solid blue line. For clarity, only the two counter-rotating streamwise vortices spanning the half-space $y \in [-1, 0]$ are represented.

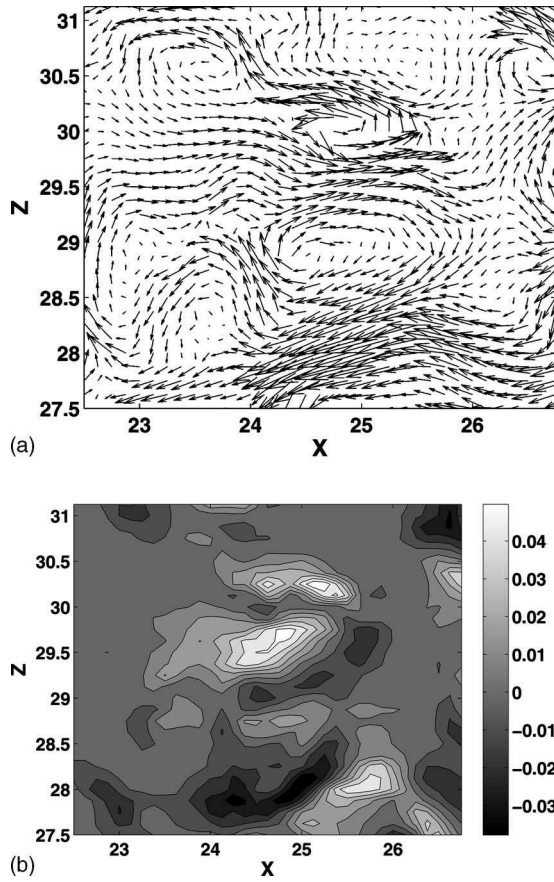


FIG. 12. (a) The flow field (U_0, W_0) depicted by vectors. Two dipoles are clearly observed. (b) The Reynolds stress U_0V_1 . There are three x -elongated patches with positive value.

these waves can be studied using this model, which will provide some elements of understanding on their nature and their role. One can already note that they contribute (albeit weakly) to the energy production because at the wing tips, the Reynolds stress U_2V_2 [in Eq. (31)] is positive, since U_2 and V_2 evolve in phase, as inferred from Fig. 7.

In conclusion, the model captures all essential features of the spot, such as its shape, its mean turbulent characteristics, and the presence of waves on the wing tips. This is an indication that further information that may be extracted from the model will be fruitful.

APPENDIX A: THE EQUATIONS FOR THE PERTURBATIONS

By expanding the velocity components $u=U(y)+u'$, $v=v'$, $w=w'$, and the pressure $p=\tilde{p}+P^0$, where P^0 is the driving pressure ($\frac{d}{dx}P^0=\text{const}$) and $(u', v', w', \tilde{p})$ are the perturbations, and inserting these expansions in Eqs. (1) and (2), we get

$$\begin{aligned} \partial_t u' + u' \partial_x u' + v' \partial_y u' + w' \partial_z u' \\ = -\partial_x \tilde{p} - U \partial_x u' - v' \frac{d}{dy} U + R^{-1} \nabla^2 u', \end{aligned}$$

$$\begin{aligned} \partial_t v' + u' \partial_x v' + v' \partial_y v' + w' \partial_z v' \\ = -\partial_y \tilde{p} - U \partial_x v' + R^{-1} \nabla^2 v', \end{aligned}$$

$$\begin{aligned} \partial_t w + u' \partial_x w' + v' \partial_y w' + w' \partial_z w' \\ = -\partial_z \tilde{p} - U \partial_x w' + R^{-1} \nabla^2 w', \end{aligned}$$

$$0 = \partial_x u' + \partial_y v' + \partial_z w'.$$

Then, using the rotational form for the nonlinearities and defining $p' = \tilde{p} + E'_{\text{loc}}$ with $E'_{\text{loc}} = \frac{1}{2}(u'^2 + v'^2 + w'^2)$, we get Eqs. (3)–(6).

APPENDIX B: THE NONLINEAR TERMS OF THE MODEL

$$\begin{aligned} N_{U_0} = & \alpha_{01} U_0 V_1 - \alpha_{06} V_1 \partial_x V_1 + \alpha_{02} U_1 V_1 \\ & + \alpha_{03} W_0 (\partial_z U_0 - \partial_x W_0) + \alpha_{04} W_1 (\partial_z U_0 - \partial_x W_0) \\ & + \alpha_{04} W_0 (\partial_z U_1 - \partial_x W_1) + \alpha_{05} W_1 (\partial_z U_1 - \partial_x W_1) \\ & + \alpha_{11} U_2 V_2 + \alpha_{12} W_2 (\partial_z U_2 - \partial_x W_2) \\ & - \alpha_{13} V_2 \partial_x V_2, \end{aligned}$$

$$\begin{aligned} N_{W_0} = & \alpha_{01} W_0 V_1 - \alpha_{06} V_1 \partial_z V_1 + \alpha_{02} W_1 V_1 \\ & + \alpha_{03} U_0 (\partial_x W_0 - \partial_z U_0) + \alpha_{04} U_1 (\partial_x W_0 - \partial_z U_0) \\ & + \alpha_{04} U_0 (\partial_x W_1 - \partial_z U_1) + \alpha_{05} U_1 (\partial_x W_1 - \partial_z U_1) \\ & + \alpha_{11} V_2 W_2 - \alpha_{13} V_2 \partial_z V_2 + \alpha_{12} U_2 (\partial_x W_2 - \partial_z U_2), \end{aligned}$$

$$\begin{aligned} N_{U_1} = & -\alpha_{07} U_0 V_1 - \alpha_{08} U_1 V_1 + \alpha_{04} W_0 (\partial_z U_0 - \partial_x W_0) \\ & + \alpha_{05} W_1 (\partial_z U_0 - \partial_x W_0) + \alpha_{05} W_0 (\partial_z U_1 - \partial_x W_1) \\ & + \alpha_{09} W_1 (\partial_x W_1 - \partial_z U_1) - \alpha_{14} U_2 V_2 \\ & + \alpha_{15} W_2 (\partial_z U_2 - \partial_x W_2) + \alpha_{16} V_2 \partial_x V_2, \end{aligned}$$

$$\begin{aligned} N_{W_1} = & -\alpha_{07} W_0 V_1 - \alpha_{08} W_1 V_1 + \alpha_{04} U_0 (\partial_x W_0 - \partial_z U_0) \\ & + \alpha_{05} U_1 (\partial_x W_0 - \partial_z U_0) + \alpha_{05} U_0 (\partial_x W_1 - \partial_z U_1) \\ & + \alpha_{09} U_1 (\partial_z U_1 - \partial_x W_1) - \alpha_{14} V_2 W_2 + \alpha_{16} V_2 \partial_z V_2 \\ & + \alpha_{15} U_2 (\partial_x W_2 - \partial_z U_2), \end{aligned}$$

$$\begin{aligned} N_{V_1} = & -\alpha_{01} U_0^2 - \alpha_{10} U_0 U_1 + \alpha_{08} U_1^2 - \alpha_{01} W_0^2 - \alpha_{10} W_0 W_1 \\ & + \alpha_{08} W_1^2 + \alpha_{06} W_0 \partial_z V_1 + \alpha_{06} U_0 \partial_x V_1 - \alpha_{17} U_2^2 \\ & - \alpha_{17} W_2^2 + \alpha_{24} W_2 \partial_z V_2 + \alpha_{24} U_2 \partial_x V_2, \end{aligned}$$

$$\begin{aligned} N_{U_2} = & \alpha_{17} U_2 V_1 + \alpha_{18} U_0 V_2 + \alpha_{19} U_1 V_2 \\ & + \alpha_{12} W_2 (\partial_z U_0 - \partial_x W_0) + \alpha_{15} W_2 (\partial_z U_1 - \partial_x W_1) \\ & + \alpha_{12} W_0 (\partial_z U_2 - \partial_x W_2) + \alpha_{15} W_1 (\partial_z U_2 - \partial_x W_2) \\ & - \alpha_{24} (V_2 \partial_x V_1 + V_1 \partial_x V_2), \end{aligned}$$

$$\begin{aligned}
N_{W_2} = & \alpha_{18}V_2W_0 + \alpha_{19}V_2W_1 + \alpha_{17}V_1W_2 - \alpha_{24}V_2\partial_zV_1 \\
& - \alpha_{24}V_1\partial_zV_2 + \alpha_{12}U_2(\partial_xW_0 - \partial_zU_0) \\
& + \alpha_{15}U_2(\partial_xW_1 - \partial_zU_1) + \alpha_{12}U_0(\partial_xW_2 - \partial_zU_2) \\
& + \alpha_{15}U_1(\partial_xW_2 - \partial_zU_2),
\end{aligned}$$

$$\begin{aligned}
N_{V_2} = & -\alpha_{25}(U_0U_2 + W_0W_2) - \alpha_{26}(U_1U_2 + W_1W_2) \\
& + \alpha_{24}(W_2\partial_zV_1 + U_2\partial_xV_1) + \alpha_{13}(W_0\partial_zV_2 + U_0\partial_xV_2) \\
& - \alpha_{16}(W_1\partial_zV_2 + U_1\partial_xV_2).
\end{aligned}$$

The coefficients of the nonlinear terms are

$$\alpha_{01} = \sqrt{35/11}/52, \quad \alpha_{02} = 45\sqrt{105/11}/52, \quad \alpha_{03} = 21\sqrt{105}/286,$$

$$\alpha_{04} = \sqrt{35}/286, \quad \alpha_{05} = 19\sqrt{105}/286, \quad \alpha_{06} = \sqrt{105}/13,$$

$$\alpha_{07} = 7\sqrt{105/11}/52, \quad \alpha_{08} = 9\sqrt{35/11}/52, \quad \alpha_{09} = 9\sqrt{35}/286,$$

$$\alpha_{10} = 19\sqrt{105/11}/26, \quad \alpha_{11} = 9\sqrt{35}/44, \quad \alpha_{12} = 5\sqrt{35/3}/22,$$

$$\alpha_{13} = 10\sqrt{105}/143, \quad \alpha_{14} = 7\sqrt{105}/44, \quad \alpha_{15} = \sqrt{35}/22,$$

$$\alpha_{16} = 15\sqrt{35}/143, \quad \alpha_{17} = \sqrt{35/11}/4, \quad \alpha_{18} = \sqrt{35}/44,$$

$$\alpha_{19} = 9\sqrt{105}/44, \quad \alpha_{24} = 5\sqrt{105/11}/26, \quad \alpha_{25} = 5\sqrt{35}/22,$$

$$\alpha_{26} = \sqrt{105}/22.$$

The coefficients of the linear terms are $b_1=8/11$, $b_2=4/(11\sqrt{3})$, $b_3=5/(11\sqrt{3})$, $b_4=9\sqrt{3}/2$, and $b_5=10/13$. Furthermore, since we use periodic boundary conditions in the numerical simulation, the mean values of the velocities are computed using the following equations, where the overbar denotes an average in the (x, z) plane:

$$\begin{aligned}
\left(\frac{d}{dt} + R^{-1}\gamma_0\right)\overline{U_0} = & -(\alpha_{01} - \alpha_{04}\beta_1)\overline{U_0V_1} \\
& -(\alpha_{02} - \alpha_{05}\beta_1)\overline{U_1V_1} \\
& -(\alpha_{11} - \alpha_{12}\beta_2)\overline{U_2V_2} - b_4\overline{U_1}/R,
\end{aligned}$$

$$\begin{aligned}
\left(\frac{d}{dt} + R^{-1}\gamma_0\right)\overline{W_0} = & -(\alpha_{01} - \alpha_{04}\beta_1)\overline{W_0V_1} \\
& -(\alpha_{02} - \alpha_{05}\beta_1)\overline{W_1V_1} \\
& -(\alpha_{11} - \alpha_{12}\beta_2)\overline{W_2V_2} - b_4\overline{W_1}/R,
\end{aligned}$$

$$\begin{aligned}
\left(\frac{d}{dt} + R^{-1}\gamma_1\right)\overline{U_1} = & (\alpha_{07} + \beta_1\alpha_{05})\overline{U_0V_1} \\
& -(\alpha_{09}\beta_1 - \alpha_{08})\overline{U_1V_1} \\
& +(\alpha_{14} + \alpha_{15}\beta_2)\overline{U_2V_2} - b_4\overline{U_0}/R,
\end{aligned}$$

$$\begin{aligned}
\left(\frac{d}{dt} + R^{-1}\gamma_1\right)\overline{W_1} = & (\alpha_{07} + \beta_1\alpha_{05})\overline{W_0V_1} \\
& -(\alpha_{09}\beta_1 - \alpha_{08})\overline{W_1V_1} \\
& +(\alpha_{14} + \alpha_{15}\beta_2)\overline{W_2V_2} - b_4\overline{W_0}/R,
\end{aligned}$$

$$\begin{aligned}
\left(\frac{d}{dt} + R^{-1}\gamma_2\right)\overline{U_2} = & (\alpha_{15}\beta_2 - \alpha_{17})\overline{U_2V_1} \\
& +(\alpha_{15}\beta_2 - \alpha_{19})\overline{U_1V_2} \\
& +(\alpha_{12}\beta_2 - \alpha_{18})\overline{U_0V_2},
\end{aligned}$$

$$\begin{aligned}
\left(\frac{d}{dt} + R^{-1}\gamma_2\right)\overline{W_2} = & (\alpha_{15}\beta_2 - \alpha_{17})\overline{W_2V_1} \\
& +(\alpha_{15}\beta_2 - \alpha_{19})\overline{W_1V_2} \\
& +(\alpha_{12}\beta_2 - \alpha_{18})\overline{W_0V_2}.
\end{aligned}$$

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